

ACOUSTIC ABSORPTION COEFFICIENT VARIATION OF SOUND ABSORBING STRUCTURES AND MATERIALS

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ABSTRACT

The task of this work is the acoustic absorption coefficient variation study of sound absorbing structures and materials. Three construction equipments have been considered according to their different cabin design. Different sound frequencies and different layers of diverse sound absorbing treatment lead to different absorption average coefficient values.

KEYWORDS: Sound, Frequency, Absorption, Coefficient, Cabin .

1. Introduction

The cabin is exposed to the noise generated by sources which are parts of the construction equipment.

$$\Delta L = 10 \lg \frac{A}{A_0} \quad [dB] \quad (1)$$

A_0 - equivalent acoustic absorbing surface of the cabin before insulating, treatment (m²AU); A - equivalent acoustic absorbing surface of the cabin after insulating treatment (m²AU)

The following relation is considered for the equivalent absorbing surface:

$$A = \sum_{i=1}^n \alpha_i S_i \quad (2)$$

S_i - the surface corresponding to "i" partial cabin area; α_i - acoustic absorbing coefficient corresponding to "i" area.

The average acoustic absorbing coefficient is calculated as shown below:

$$\alpha_m = \frac{\sum \alpha_i S_i}{S} \quad (3)$$

Acoustic absorbing inside the cabin it is also characterized by the absorption constant "R"[2], [3]:

$$R = \frac{\alpha_m S}{1 - \alpha_m} \quad (4)$$

2. Case study

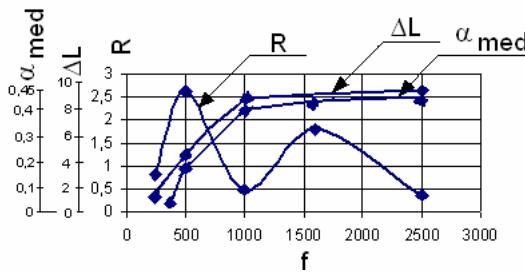
2.1. MMT 45 equipment

The following values have been considered for MMT 45 equipment cabin:

$$S_1 = 4.7 m^2, S_2 = 1.7 m^2, S_3 = 3.8 m^2.$$

Table no. 1
 Four layers

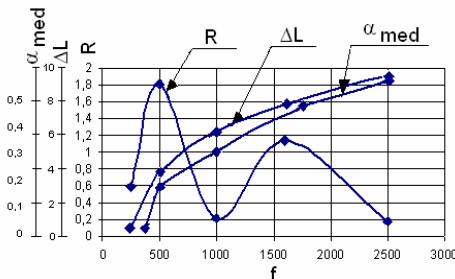
f	α	α_{med}	ΔL	R
250	0,1	0,071	0,61	0,775
500	0,4	0,209	5,318	2,692
1000	0,8	0,393	8,066	0,502
1600	0,88	0,43	8,455	2,031
2500	0,92	0,448	8,637	0,408

Figure 1- Plotting $R, \Delta L, \alpha_{med}$ function f for four layers for MMT 45 equipmentTable no. 4
One layer

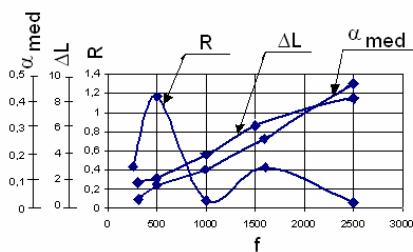
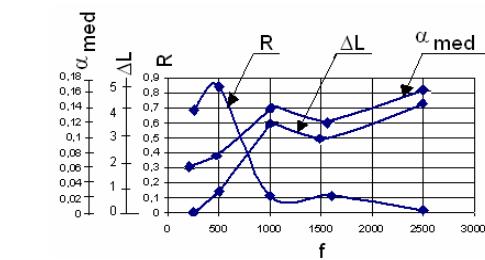
f	α	α_{med}	ΔL	R
250	0,08	0,061	0	0,667
500	0,11	0,075	0,882	0,829
1000	0,26	0,144	3,713	0,112
1600	0,21	0,121	2,958	0,114
2500	0,31	0,167	4,357	0,023

Table no. 2
Three layers

f	α	α_{med}	ΔL	R
250	0,06	0,052	0,71	0,561
500	0,28	0,154	3,982	1,85
1000	0,54	0,273	6,487	0,211
1600	0,83	0,407	8,216	1,27
2500	0,99	0,481	8,939	0,195

Figure 2 - Plotting $R, \Delta L, \alpha_{med}$ function f for three layers for MMT 45 equipmentTable no. 3
Two layers

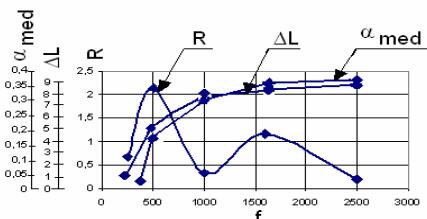
f	α	α_{med}	ΔL	R
250	0,03	0,038	2,04	0,407
500	0,17	0,103	2,242	1,169
1000	0,29	0,158	4,111	0,076
1600	0,56	0,283	6,631	0,46
2500	1	0,485	8,98	0,072

Figure 3 - Plotting $R, \Delta L, \alpha_{med}$ function f for two layers for MMT 45 equipmentFigure 4 - Plotting $R, \Delta L, \alpha_{med}$ function f for one layer for MMT 45 equipment**2.2. S 1201 excavator**

For S 1201 excavator the following values

cabin: $S_1 = 3.8m^2$; $S_2 = 1.2m^2$; $S_3 = 511m^2$.Table no. 5
Four layers

f	α	α_{med}	ΔL	R
250	0,1	0,062	0,56	0,673
500	0,4	0,174	5,037	2,145
1000	0,8	0,323	7,728	0,322
1600	0,88	0,353	8,111	1,171
2500	0,92	0,368	8,291	0,187

Figure 5 - Plotting $R, \Delta L, \alpha_{med}$ function f for four layers for S 1201 excavatorTable no. 6
Three layers

f	α	α_{med}	ΔL	R
250	0,06	0,047	0,64	0,503
500	0,28	0,129	3,745	1,511
1000	0,54	0,226	6,178	0,147
1600	0,83	0,335	7,876	0,759
2500	0,99	0,394	8,589	0,096

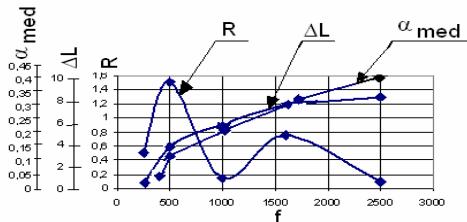


Figure 6 - Plotting $R, \Delta L, \alpha_{med}$ function f for three layers for S 1201 excavator

Table no. 7
Two layers

f	α	α_{med}	ΔL	R
250	0,03	0,036	1,82	0,379
500	0,17	0,088	2,084	0,984
1000	0,29	0,133	3,868	0,058
1600	0,56	0,234	6,319	0,3
2500	1	0,398	8,63	0,038

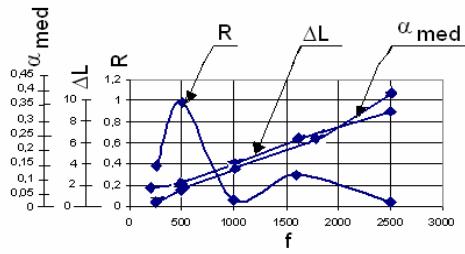


Figure 7 Plotting $R, \Delta L, \alpha_{med}$ function f for two layers for S 1201 excavator

Table no. 8
One layer

f	α	α_{med}	ΔL	R
250	0,08	0,055	0	0,587
500	0,11	0,066	0,811	0,717
1000	0,26	0,122	3,486	0,081
1600	0,21	0,103	2,763	0,082
2500	0,31	0,14	4,105	0,013

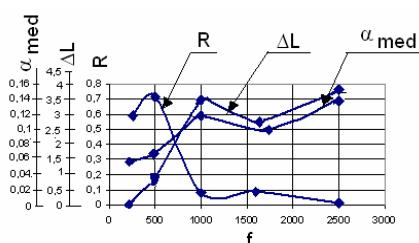


Figure 8 Plotting $R, \Delta L, \alpha_{med}$ function f for one layer for S 1201 excavator

2.3 CVA 10 vibrating compactor

Considered value: $S_1 = 4,5 m^2; S_2 = 2,1 m^2$

$$S_3 = 2,2 m^2.$$

Table no. 9
Four layers

f	α	α_{med}	ΔL	R
250	0,1	0,07	0,57	0,787
500	0,4	0,2	5,109	2,603
1000	0,8	0,373	7,815	0,469
1600	0,88	0,408	8,2	1,793
2500	0,92	0,425	8,381	0,347

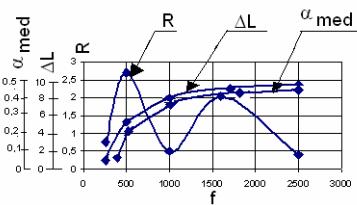


Figure 9 - Plotting $R, \Delta L, \alpha_{med}$ function f for four layers for CVA 10 vibrating compactor

Table no. 10
Three layers

f	α	α_{med}	ΔL	R
250	0,06	0,053	0,66	0,583
500	0,28	0,148	3,805	1,81
1000	0,54	0,261	6,258	0,206
1600	0,83	0,386	7,964	1,139
2500	0,99	0,455	8,68	0,172

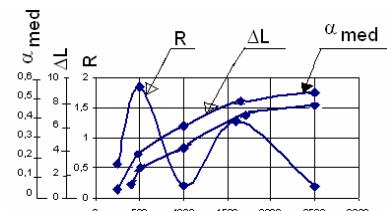


Figure 10 - Plotting $R, \Delta L, \alpha_{med}$ function f

Table no. 11
Two layers

f	α	α_{med}	ΔL	R
250	0,03	0,04	1,87	0,434
500	0,17	0,101	2,124	1,164
1000	0,29	0,153	3,93	0,078
1600	0,56	0,269	6,399	0,429
2500	1	0,46	8,721	0,067

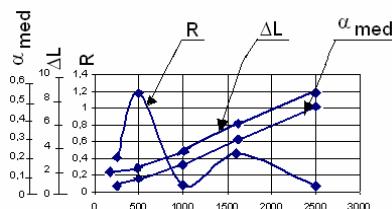


Figure 11 - Plotting $R, \Delta L, \alpha_{med}$ function f for two layers for CVA 10 vibrating compactor

Table no. 12
One layer

f	α	α_{med}	ΔL	R
250	0,08	0,062	0	0,684
500	0,11	0,075	0,829	0,84
1000	0,26	0,14	3,544	0,111
1600	0,21	0,118	2,813	0,112
2500	0,31	0,161	4,17	0,021

S_1 - sound absorbing treated geometric surface; S_2 - sound absorbing untreated metallic surface; S_3 - glass surface.

3. Conclusions

A study of acoustic absorption coefficient variation for sound absorbing structures and materials considering three construction equipments has been performed. The sound material has been considered from one to four layers [1]. Different values have been determined for: R -absorption constant, ΔL - global noise reducing, α_{med} - sound absorbing average coefficient. Different frequencies have been considered. The results can be compared considering the tables below[4]. Notations in the tables above: FB – very good, C – onstant, B – good, S – low.

Table no. 13

Equip- ment	Para- metes	Frequency				
		250	500	1000	1600	2500
MMT 40	R	FB	S	B	FB	B
	ΔL	FB	B	C	C	C
	α_{med}	FB	B	C	C	C
S 1201	R	FB	B	B	FB	B
	ΔL	FB	B	C	C	C
	α_{med}	FB	B	C	C	C
CVA 10	R	FB	S	B	FB	B
	ΔL	FB	B	C	C	C
	α_{med}	FB	B	C	C	C

Table no. 14

Equip- ment	Para- metes	Frequency				
		250	500	1000	1600	2500
MMT 40	R	FB	S	B	FB	B
	ΔL	C	B	C	C	C
	α_{med}	C	B	C	C	C
S 1201	R	FB	S	B	FB	B
	ΔL	FB	FB	B	C	C
	α_{med}	FB	FB	B	B	B
CVA 10	R	FB	S	B	FB	B
	ΔL	FB	FB	B	B	B
	α_{med}	FB	FB	B	B	B

Table no. 15

Equip- ment	Para- metes	Frequency				
		250	500	100 0	160 0	250 0
MMT 40	R	B	S	B	S	B
	ΔL	FB	B	B	S	S
	α_{med}	FB	B	B	S	B
S 1201	R	B	S	B	S	B
	ΔL	FB	B	C	C	C
	α_{med}	FB	B	C	C	C
CVA 10	R	FB	S	B	FB	B
	ΔL	FB	B	C	C	C
	α_{med}	FB	B	C	C	C

Table no. 16

Equip- ment	Para- metes	Frequency				
		250	500	1000	1600	2500
MMT 40	R	B	B	S	B	S
	ΔL	S	B	S	S	B
	α_{med}	S	B	S	S	B
S 1201	R	B	B	S	B	S
	ΔL	S	B	S	S	FB
	α_{med}	S	B	S	S	B
CVA 10	R	B	B	S	B	S
	ΔL	S	B	B	FB	B
	α_{med}	B	B	B	FB	B

REFERENCES

- [1] Anghelache D. – *The phonic isolation of the S1201 Excavator cabin* - "Romanian Journal of Acoustics and Vibration" editat de "Romanian Society of Acoustics", vol. IV, nr. 2/2007, ISSN 1584-7284.
- [2] Bratu, P. – *Acustica interioară pentru construcții și mașini*, Editura Impuls, București, 2002
- [3] Iudin, Ia – *Izolarea împotriva zgomotelor*, Editura Tehnică, București, 1968
- [4] *** Directive 2002/44/EC